STANDARD IB CLUTCH IB214P

QUALITY IS STANDARD

- AVAILABLE IN SIZED 11.5" THRU 21.0"
- TAPERED ROLLER MAIN BEARINGS
- OPTIONAL SINTERED IRON PLATES
- · OPTIONAL BALL BEARING THROW OUT
- BUILT IN HEX NUT
- · CREATES SUITABLE APPLICATION TORQUE CAPACITY
- MORE SUITABLE FOR SIDE LOAD APPLICATIONS
- CREATES 25% HIGHER TORQUE CAPACITY
- ALLOWS FOR MORE FREQUENT ENGAGEMENS
- EASES ADJUSTMENT VERIFICATION



SPECIFICATIONS - IB214P0, IB214P1

Model Number	SAE HSG.	Dimension "A" mm (in)	Max. Input Torque Nm (lb-ft)		Maximum Safe Speed				
					Solid Plates		Split Plates		Weight
			Organic	Sintered	Cast Drive Ring	Nodular Drive Ring	Cast Drive Ring	Nodular Drive Ring	kg (lbs)
IB214P0	0	100 (3.937)	2198 (1620)	2748 (2025)	2500	3000	1950	2750	150 (328)
IB214P1	1	80 (3.1496)							

LOAD CLASSIFICATIONS BASED UPON AGMA LOAD CHARACTERISTICS

DDIME MOVED	DURATION	DRIVEN MACHINE LOAD CLASSIFICATIONS				
PRIME MOVER	OF SERVICE	UNIFORM	MODERATE SHOCK	HEAVY SHOCK		
Electric motor	Up to 3 hours per day	1.00	1.25	1.50		
	3-10 hours per day	1.00	1.25	1.75		
	Over 10 hours per day	1.25	1.50	2.00		
Multi-cylinder internal combustion engine	Up to 3 hours per day	1.00	1.25	1.75		
	3-10 hours per day	1.25	1.50	2.00		
	Over 10 hours per day	1.50	1.75	2.25		
Multi-cylinder internal combustion engine with high torque rise	Up to 3 hours per day	1.50	1.75	2.25		
	3-10 hours per day	1.75	2.00	2.50		
	Over 10 hours per day	2.00	2.25	2.75		
Single cylinder internal combustion engine	Up to 3 hours per day	1.25	1.50	2.00		
	3-10 hours per day	1.50	1.75	2.25		
	Over 10 hours per day	1.75	2.00	2.50		

All clutch engagements to be with prime mover below 1000 RPM. High inertia loads may require use of larger clutch. Contact Twin Disc application engineering department for assistance.

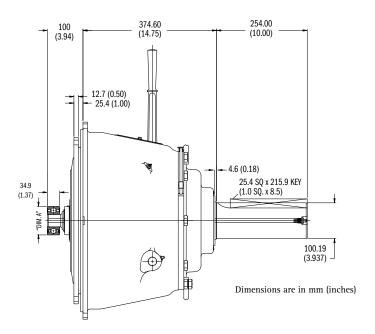
TO CALCULATE APPLICATION TORQUE:

 $\frac{5252 \text{ x HP}}{\text{Engine RPM}} = \text{Torque}$

Torque x Load Factor = Application Torque
Use load factor from chart at left



IB214P0



IB214P0 & IB214P1 - ALLOWABLE SIDE LOAD, KG (LBS)

IDET II O G IDET II T ALLOWADEL OIDE EORD) NG (EDO)									
PTO MODEL	RPM	X DISTANCE, mm (in)							
		25.4 (1.0)	50.8 (2.0)	76.2 (3.0)	101.6 (4.0)	127.0 (5.0)	152.4 (6.0)	177.8 (7.0)	
	1000	3629 (8000)	3425 (7550)	3175 (7000)	000E (E07E)				
IB214P0	1200	3425 (7550)	3243 (7150)	3084 (6800)	2665 (5875)	2279 (5025)	1826 (4025)	1826 (4025)	
M2137	1800	3039 (6700)	2869 (6325)	2722 (6000)	2201 (E2E0)				
	2400	2790 (6150)	2631 (5800)	2495 (5500)	2381 (5250)				
	1000								
IB214P1	1200	2989 (6590)	2240 (5100)	1000 (4050)	1000 (2000)	1400 (2120)	1050 (0700)	1120 (2470)	
M1985	1800		2340 (5160)	1928 (4250)	1633 (3600)	1420 (3130)	1252 (2760)	1120 (2470)	
	2400	2790 (6150)							

The following general formula should be used for determining the actual applied load: $L = \frac{126,000 \text{ x HP}}{\text{N x D}} \text{ x F x LF}$

WHERE L = Actual Applied Load (lbs)

N = Shaft Speed (RPM)

D = Pitch Diameter (in) of Sheave, etc.

F = Load Factor

1.0 for Chain or Gear Drive, 1.5 for Timing Belts, 2.5 for All V Belts, 3.5 for Flat Belts

LF = 2.1 for Reciprocating Compressors and other Severe Shock Drives and 1.8 for Large Inertia

Type Drives (i.e. crushers, chippers, planers, etc.)

Compound Drives and Power Engaged Power Take-Off applications must have written factory review.

Twin Disc, Incorporated reminds users of these products that their safe operation depends on use in compliance with engineering information provided in our catalog. Users are also reminded that safe operation depends on proper installation, operation and routine maintenance and inspection under prevailing conditions. It is the responsibility of users (and not Twin Disc, Incorporated) to provide and install guards or safety devices which may be required by recognized safety standards or by the Occupational Safety and Health Act of 1970 and its subsequent provisions.

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